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EVALUATION OF CV 100 FUEL COMBUSTION CATALYST ONBOARD THE TUGBOAT BRAHMA

Prepared for Tidewater Marine by UHI Corporation in cooperation with Tidewater Marine

August 17, 1982

INTRODUCTION

Arrangements were made between Tidewater Marine and UHI Corporation to conduct an exhaust gas analysis and engine performance (fuel economy) study onboard an inshore tug. The purpose of the study was to document the changes in exhaust gas composition and in fuel economy resulting from the addition of CV 100 Fuel Combustion Catalyst to the No. 2 diesel fueling a boat's main engines.

The <u>Brahma</u> was selected as a test boat, representative of the inshore tug fleet. A UHI technician with analytical equipment boarded the <u>Brahma</u> on July 23, 1982, and collected engine performance data (exhaust emissions) using untreated fuel. On August 13, 1982, duplicate procedures were followed and data collected with treated fuel.

Analytical Equipment:Sun Electric MGA-90 Multiple Gas AnalyzerVessel Specification:
Name:Main Engines:2 GM Model V871Class:600 HP Twin Screw Supply TugMaximum HP:700 @ 2300 RPMClass:600 HP Twin Screw Supply TugMaximum HP:700 @ 2300 RPMLength:70 feetContinuous HP:460 @ 1800 RPMBeam:26 feetGears:Twin Disc 4.5:1Load Draft:6 feetPropellers:54" Dia x 34"-Pitch BladeLight Draft:5 feetSpeed:9 MPHFuel Consumption:30 Gal/Hr.30 Gal/Hr.30 Gal/Hr.

METHODOLOGY

Tom Medynski and Dan Gaiennie, engineers for Tidewater Marine, and Craig Flinders, UHI Technician, conducted baseline testing onboard the tugboat <u>Brahma</u> on July 23, 1982. Using the Sun Electric MGA-90 four gas analyzer, baseline exhaust gas levels of carbon monoxide (CO), carbon dioxide (CO₂), unburned hydrocarbons (HC) and oxygen (O₂), were recorded using the following procedure.

- 1) Starboard and port main engines brought up to temperature.
- 2) Exhaust readings taken at 1700, 1500 and 1200 RPM's while pushing against the dock to simulate load of working conditions.
- Exhaust readings taken at 1700, 1500 and 1200 RPM's while traveling 3) downriver to simulate load of cruise conditions.
- 4) Exhaust readings taken at 1700, 1500 and 1200 RPM's while traveling upriver to simulate load of cruise conditions against the current.

After the baseline readings were taken, number one and two fuel tanks, and the day tank were treated with CV 100 at a ratio of one gallon of CV 100 to 1600 gallons of fuel. The Brahma ran for 107 hours with treated fuel before Tidewater and UHI personnel returned to record test data. Readings (with CV 100) were then taken using the same procedures as baseline. Baseline data was read from the MGA-90 meters by both Tidewater and UHI personnel. The "treated" portion of the tests was recorded by Dan Gaiennie of Tidewater.

It should be noted that the MGA-90 analyzers are intended for use with gasoline engines. Application to measuring diesel engine exhaust could introduce some error due to the low scale readings observed and possible particulate interference.

DATA ANALYSIS

The emissions data may be used to determine relative efficiency values using a carbon balance for a given engine if certain assumptions are made. If it is assumed that the steady state engine operating point, (torque and speed), and ambient conditions remain unchanged from one test to another, then exhaust emissions data can be used to indicate an efficiency change due to modifications.

Since the exhaust mass flow is not available in a field test of this kind, absolute values of fuel efficiency cannot be determined using the carbon balance technique. Therefore, a "performance factor" can be calculated for each test which is based on the above assumptions and is related to fuel efficiency.

The performance factor may be calculated according to the following relation:

2553.5

Performance Factor = $\frac{20000}{[0.86 \text{ (ppmHC)} + 0.429 \text{ (Vol%CO)} + 0.273 \text{ (Vol%CO}_2)]}$

OBSERVATIONS

FILTERS

The MGA-90 is equipped with two outside filters that collect particulates which prevents blockage: of hoses, filters, and analyzers inside the unit. UHI technicians added an additional 25 micron filter to the sampling hose to assist the other 49 micron filters. During the baseline tests, one on-line filter (previously used in other testing) was plugged by particulates. A new filter was installed, replacing the plugged filter; mid-way through the rest of the baseline testing, this filter plugged also. The baseline was finished without a filter on the sampling hose.

During the CV 100 tests a new filter was installed and used throughout the balance of the study. Unlike the baseline tests, it remained free of carbon particulate plugging and "black discoloration".

EXHAUST PLUME

UHI and Tidewater Marine engineers, observed a heavy black exhaust plume during baseline evaluations. Accompanying those exhaust emissions, was a black particulate that blew into the atmosphere, falling to rest on the ship as well as surrounding areas.

Followup testing showed drastic reductions in visual exhaust content upon completion of a 107 hour test using CV 100. The particulate observed during the baseline was greatly diminished along with a reduction in harmful exhaust emissions.

The heavy black exhaust plume had diminished with CV 100 to a grey, less dense cloud.

CONCLUSIONS

The carbon balance technique used in this report is recognized as one of the most accurate methods of assessing fuel economy when applied under controlled conditions. It is based upon changes in exhaust emissions concentrations at a specific load and with a certain fuel type. The specific reductions in carbon containing exhaust constituents, as recorded by Tidewater Marine engineers, demonstrates an improvement in fuel consumption in the two 8V-71 engines powering the Brahma while using CV 100.

The performance factors, (with qualifying assumptions), also indicate an improvement in fuel economy with CV 100. The percentage change in fuel consumption of CV 100 treated fuel over untreated fuel is in excess of thirty percent with uncertainty at all RPM levels. This unusually high fuel economy improvement may, to some extent, be a result of variables in load that occur in field testing, and a normal drift with NDIR analyzers, (MGA-90), could have occurred. Also, the HC and CO concentrations from the diesel were measured on the extreme low end of the analyzer scales which could give rise to both recording and detector errors. These uncertainties have not been quantified, so absolute accuracies cannot be determined. It is likely that the mechanical and calculated results show a real trend in improved fuel economy with CV 100.

The levels of all harmful emissions were reduced; unburned hydrocarbons and carbon monoxide were reduced to nearly half of untreated levels with CV 100. Smoke and particulates were also reduced, based on visual observations made by UHI and Tidewater personnel.

The performance factors may be scaled to be closer to actual MPG values by a common factor which is based on dynamometer exhaust flow and temperatures data and fuel specific gravity and carbon to hydrogen ratio. Each performance factor was scaled by the same coefficient for a given test type so that percentage changes were unaffected.

Table 1 summarizes the emissions data from the tugboat field tests work in the calculations for relative efficiency. Dynamometer data for the 8V-71 Detroit Diesel are shown in Table 2; from which the constant scaling factor was determined. The scaled performance factors for each RPM and test type with and without CV 100 are listed on Table 3.

TABLE 1 Emissions Data

DATA PAGE	MODE	RPM	<u>CV 100</u>	<u>CO28</u>	028	<u>C0</u> %	HC_ppm
1	Pushing Starbd	1700	No	4.73	12	0.053	21.5
2	Pushing Port	1700	No	4.33	12.2	0.188	19
3	Pushing Starbd	1700	Yes	3.0	13.4	0.06	14.33
3	Pushing Port	1700	Yes	2.9	13.4	0.143	33%
4	Pushing Starbd	1500	No	3.17	15.0	0.023	20
5	Pushing Port	1500	No	2.97	15.3	0.015	20
6	Pushing Starbd	1500	Yes	1.93	15.3	0 25	15
6	Pushing Port	1500	Yes	1.83	15.3	0	15
7 7 8 8	Pushing Starbd Pushing Port Pushing Starbd Pushing Port	1200 1200 1200 1200	No No Yes Yes	2.33 2.0 1.08 1.1	16.8 17.3 17.9 17.8	0.01 0.01 0 30	$ \begin{array}{c} 19.3 \\ 19.0 \\ 12.8 \\ 13.2 \end{array} $
9 9 10 10	Upriver Port Upriver Starbd Upriver Port Upriver Starbd	1700 1700 1700 1700	No No Yes Yes	3.22 3.04 1.51 1.2	014.97 15.1 17.3 18.1	0 0 0 0 3 ⁰ /.	20 20 11.3 11.2
11	Upriver Port	1500	No	2.06	016.5	0	20
11	Upriver Starbd	1500	No	1.9	17.3	0	20
12	Upriver Port	1500	Yes	0.94	18.8	0	11
12	Upriver Starbd	1500	Yes	0.8	19.2	0	11
13 13	Upriver Port Upriver Starbd Upriver Upriver	1200 1200 1200 1200	No No Yes Yes		18.0 17.75 DATA DATA	0 0	19.5 20.0
14 14 15 15	Downriver Port Downriver Starbd Downriver Port Downriver Starbd	1700 1700 1700 1700	No No Yes Yes	2.9 2.79 1.53 1.29	15.4 15.5 17.3 17.8	0 0 0 0 0 0	$\begin{pmatrix} 20\\ 20\\ 11\\ 11 \end{pmatrix}$ $+3'b$
16	Downriver Port	1500	No	2.3	016.5	0	19
16	Downriver Starbd	1500	No	2.25	16.4	0	19.5
17	Downriver Port	1500	Yes	1.1	18.6	0	10.9
17	Downriver Starbd	1500	Yes	0.86	18.9	0	10.8
18	Downriver Port	1200	No	1.56	17.6	0	19
18	Downriver Starbd	1200	No	1.66	17.3	0	18.2
19	Downriver Port	1200	Yes	0.71	19.3	0	11.3
19	Downriver Starbd	1200	Yes	0.6	19.5	0	11.0

TABLE II

Detroit Diesel Dynamometer Studies Under Load 8V-71 Exhaust Flow and Temperature Data

	1200 RPM	1500 RPM	1700 RPM
Exhaust Temperature	760° F	770° F	775° F
Intake Airflow	510 CFM	630 CFM	710 CFM
Exhaust Flow	1140 CFM	1420 CFM	1610 CFM

Cubic Inch Displacement - 568

Mode	RPM	CV 100	Performance Factor*
PS	1700	No	1.247 2501
PS	1700	Yes	1.24 3 35 %
PP	1700	No	
PP	1700	Yes	1.34 330%
PS	1500	No	1.72 35% 2.64 3 35%
PS	1500	Yes	2.64
PP	1500	No	1.80 3 34%
PP	1500	Yes	2.72) 51 (6
PS	1200	No	2.20 2 35%
PS	1200	Yes	3.41 5 55
PP	1200	No	2.43 3 39%
PP	1200	Yes	3.97 >
US	1700	No	1.74 2 55%
US	1700	Yes	3.86
UP	1700	No	1.68 3.35 50%
UP	1700	Yes	3.35
US	1500	No	2.36 3 52%
US	1500	Yes	4.51
UP	1500	No	2.26 3 50%
UP 📉	1500	Yes	4.54 5 50 10
DS	1700	No	1.84 2 50°lo
DS	1700	Yes	3.72 5
DP	1700	No	1.80 2 46%
DP	1700	Yes	3.35 3 76 6
DS	1500	No	2.17 2 54%
DS	1500	Yes	4.79
DP	1500	No	2.16 7 48%
DP	1500	Yes	4.19
DS	1200	No	2.75 3 52%
DS	1200	Yes	5.79 5
DP	1200	No	2.78 7 47%
DP	1200	Yes	5.30 5 9 6

TABLE III Scaled Performance Factors

* Performance Factor is roughly same as miles/gallon. See list of assumptions for further information.

APPENDIX A

SUMMARY REVIEW - EFFECT OF CV 100 IN INTERNAL COMBUSTION ENGINES

Background

The combustion related efficiency of an internal combustion engine may be increased in two independent ways. First, the hydrocarbon fuel may be more completely burned in the combustion chamber, with a theoretical exhaust gas consisting of carbon dioxide, water vapor and nitrogen. Actual engine exhaust contains small concentrations of carbon monoxide and unburned hydrocarbons as well as other products of incomplete combustion and molecular dissociation. For a typical engine, the conversion of carbon monoxide to carbon dioxide, and conversion of unburned fuel to water vapor and carbon dioxide, represents a potential improvement in combustion efficiency of up to approximately 3%.

Second, the combustion process may occur completely when the piston is at or just past top dead center, corresponding to minimum combustion chamber volume in a spark ignition engine. Thus, the combustion energy is released to obtain maximum combustion pressure rise for greatest contribution to crankshaft torque. (The ideal cycle theorizes combustion heat release at constant pressure as the piston moves downward, due to the interval during which fuel is injected into the combustion chamber.) Because combustion requires a finite time period for preflame reactions, ignition and flame propagation, the actual combustion related engine efficiency is reduced from the hypothetical ideal case which models these processes as occurring instantaneously. A reduction in combustion losses without destructive rates of pressure rise results in improved efficiency as the engine operates more closely to the ideal Otto (spark ignition) or Diesel thermodynamic cycle.

CV 100 Effect

The presence of CV 100 in a hydrocarbon fuel seems to affect both the completeness of combustion and the time required for combustion. While detailed combustion studies with CV 100 have not been conducted to date, a model of the CV 100 mode of action which fits observed data may be formulated.

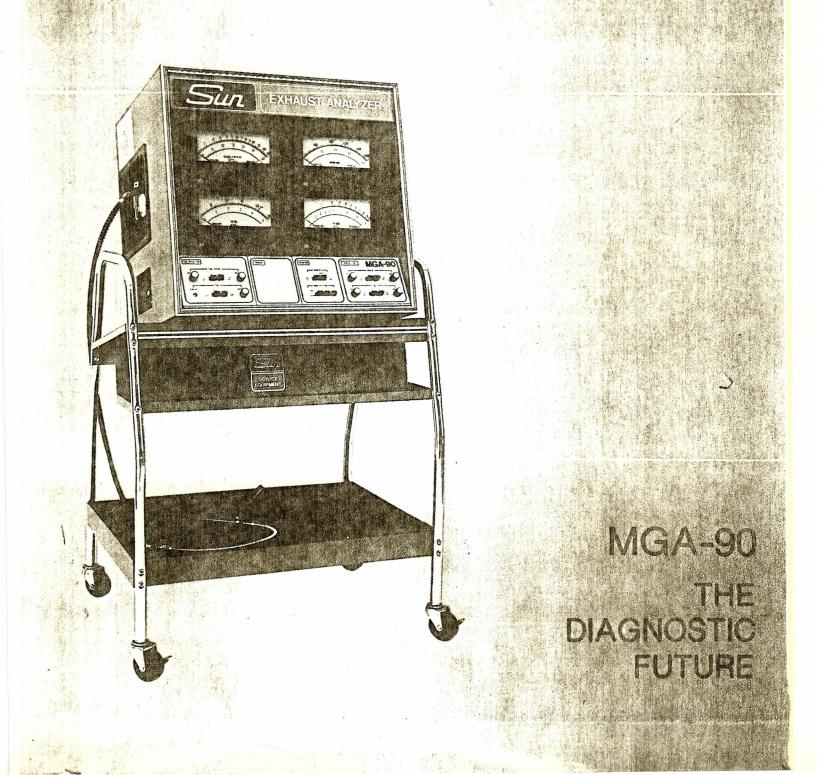
CV 100 has an affect on the amount of fuel which can be burned in a given time period in an engine cylinder. Thermal decomposition of the ferrous picrate molecules in CV 100 in the combustion chamber provides catalytic iron surfaces and picrate molecules which become available to act as localized ignition sources. Assuming that the CV 100 is uniformly dispersed in the fuel/air mixture, the mixture becomes subject to a multitude of ignition sources after the initial expanding flame kernel begins to propagate from the spark plug tip.

The resulting inflammation is more uniform than that produced by a single propagating flame front, thus increasing the amount of fuel burning when the piston is near top dead center. This effectively decreases the time required for combustion of all the fuel in the compressed mixture, which increases the energy conversion efficiency of the processes.

The consistent reduction in carbon monoxide emissions with CV 100 in automobiles, an effect which is expected with the ferrous form of the iron ion in CV 100, indicated a potentially measurable increase in combustion efficiency. The unburned hydrocarbon fuel may be reduced only slightly with CV 100 but combustion efficiency can still increase due to a strong dependence on carbon monoxide conversion to carbon dioxide,

APPENDIX B





BID DESCRIPTION

The MGA-90 Multiple Gas Analyzer is a solid-state, non-dispersive infrared exhaust analyzer with four (4) 8-inch (20 cm) analog meters. This power sampling system is designed to measure hydrocarbons (HC), carbon monoxide (CO), oxygen (O₂) and carbon dioxide (CO₂) in the exhaust of gasoline-driven internal combustion engines. It also provides an analog readout of rpm from 0-3000 rpm in 10 rpm increments. Gas calibrated with automated leak detection. The MGA-90 includes a PTS-33 test stand. It can also be mounted overhead from a rail using an optional head suspension kit.

California Bureau of Automotive Repair approved.

Specify MGA-90 Multiple Gas Analyzer or equivalent.

SPECIFICATIONS

Meters — Four, 8-inch (20 cm), fluorescent backlighted, D'Arsonval-jeweled bearing movement, tamperproof with 400% overload protection. All scales have indicator lights.

%O2 Meter - Two scales: 0-5%; 0-25%.

PPM HC Meter — Two scales: 0-400 ppm; 0-2000 ppm.

%CO Meter - Two scales: 0-2%; 0-10%.

%**CO₂/RPM Meter** — Two scales: 0-16% CO₂; 0-3000 rpm.

Tester Controls — Color-coded pushbutton, switch and potentiometer controls for total front panel control of all functions.

Warm-Up — 15 minutes @ 70°-110°F; 20 minutes @ 35°F for total system readiness.

System Accuracy HC, CO, O₂ & CO₂: ± 3% of full scale.

Temperature Range — 35° to 120° F operating; -20° to 130° F storage.

Relative Humidity - Up to 85% non-condensing.

Interference Effects — Less than ± 10 ppm HC or ± .05% CO of full scale (low range).

System Response Time — 8 seconds for 90% response of full scale.

Drift - ± 3% of full scale in one hour.

Repeatability - Less than ± 2% of full scale.

Power - 115V ac, 60 Hz @ 4.5 A.

Circuit Protection — Circuit breaker protection provided for the main console and infrared.

Dimensions - 24.5" Lx30.5" Hx18" D (62 x77 x46 cm)

Weight - 100 lbs. (45 kg).

Construction — All metal housing painted with blue textured and Sun Red enamel, resistant to oil, gaso-line, chipping and scratching.

ACCESSORIES INCLUDED

Stainless steel exhaust pickup probe and 25 feet (8m) of sample hose.

Anti-Dilution probe adapter.

Standard test lead with clamp-on tachometer connector.

Calibration gas bottle and mounting.

Literature including Operators Manual and Quick Reference Guide.



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FIGURE I

CARBON BALANCE TECHNIQUE

ASSUMPTIONS: C₈H₁₅ and SG = 0.78 Time is constant Load is constant RPM is constant

DATA:

$$pf1 = Calculated Performance Factor (Baseline)$$

 $pf2 = Calculated Performance Factor (Treated)$
 $PF1 = Performance Factor (adjusted for Baseline exhaust mass)$
 $PF2 = Performance Factor (adjusted for Treated exhaust mass)$
 $T = Temperature (F^{\circ})$
 $F = Flow (exhaust CFM)$
 $SG = Specific Gravity$
 $VF = Volume Fraction$
 $VFCO_2 = "reading" \div 100$
 $VFO_2 = "reading" \div 100$
 $VFHC = "reading" \div 1,000,000$

EQUATIONS:

 $1wt = (VFHC)(86) + (VFCO)(28) + (VFCO_2)(44) + (VFO_2)(32) + [(1-VFHC-VFCO-VFO_2-VFCO_2)(28)]$

VFCO = "reading" \div 100

pf1 or pf2 = $\frac{2952.3 \times Mwt}{86(VFHC)+13.89(VFCO)+13.89(VFCO_2)}$

PF1 or PF2 = $\frac{\text{pf x (T+460)}}{\text{F}}$

PERCENT INCREASE (OR DECREASE IN FUEL ECONOMY = $\left(\frac{PF_1 - PF_2}{PF_1}\right) \times 100$

Germane - Page Two

5. The more correct version is based on the mass functions of exhaust constituents which are obtained by assuming that the exhaust is composed entirely of unburned HC, CO, CO_2 , O_2 and N_2 .

mass fraction; = volume fraction; $(\frac{\text{molecular weight}}{\text{molecular weight of total mixture}})$

 $MW_{t} = Molecular wt. of total mixture = (volfracHC)(86)+(volfracCO)(28) + (volfracCO_{2})(44)+(volfracO_{2})(32)+(1-VF_{HC}-VF_{CO}-VF_{CO_{2}}-VF_{O_{2}})(28)$

6. The equation becomes :

III. P.F.=
$$(\frac{3785 \text{ SG}}{y} \frac{x+1}{y} \frac{1}{0.0833 \frac{x+1}{y}} \frac{(\frac{86}{MW_t})(\text{ppmHC}) + 0.429(\frac{28}{MW_t})(\text{volfracCO}) + 0.273(\frac{44}{MW_t})(\text{volfracCO}_2)}{y}$$

One only needs to know mass of exhaust per mile to obtain miles per gallon. This may be calculated based on certain assumptions concerning exhaust flow and density. If the mass of exhaust per mile is not calculated, the performance factor is:

$$[V.P.F.=(0.0833 \times +1) [\frac{86}{0.0833 \times +1} (ppmHC)+(12.01) (volfracCO)+(volfracCO_2)]]$$

7. As mentioned above: (P.F.) $\times (Mass exhaust/mi) = MPG$

Mass exhaust/mi= (velocity exhaust) (area of pipe) (density of exhaust)

mi.

miles traveled=(miles/min)(minutes of test)

where miles/min is obtained from equivalent vehicle velocity for a given engine speed.

8. Equation IV. is most technically correct for comparison purposes.

NOTE: Tidewater calculations based on equation II. with C_8H_{15} fuel and SG=0.78. This gives:

P.F.= 2553.5 0.86 (ppmHC)+0.429 (volfracCO)+0.273 (volfracCO₂)

NOTE: Time of measurement must be consistent.

Dr. G.J. Germane January, 1983

DERIVATION OF CARBON BALANCE "PERFORMANCE FACTOR"

1. Based on carbon balance fuel economy:

F.E. = $\frac{P_{cf}}{M_{ce}/D}$ Where: $\frac{P_{cf}}{M_{ce}}$ is more carbon per volume of fuel (gm/gal) M_{ce} is mass of carbon in exhaust gas (gm) D is distance traveled (miles)

2. Assume - mass of carbon in fuel is same as mass of carbon in exhaust, and define fuel composition to be $C_V H_X$

F.E. =
$$\left(\frac{12y}{x + 12y}\right) \stackrel{PC_{yH_{x}}}{=} \frac{12y}{x + 12y} (gm/mi HC) + (\frac{12}{28}) (gm/mi CO) + \frac{12}{44} (gm/mi CO_{2})$$

Where:
$$P_{C_yH_x}$$
 = (fuel specific gravity) x (density of water)
Which equals (3785)(SG of fuel)(gm/gal)
also, $\frac{12y}{x + 12y}$ is written as $\frac{1}{0.0833 \frac{x}{y} + 1}$

3. The working formula for fuel economy then becomes:

1. F.E.=
$$\frac{3785 \text{ SG}}{(0.0833 \text{ x}+1)[\frac{1}{0.0833 \text{ x}+1}} (\text{gpmHC})+0.429(\text{gpmCO})+0.273(\text{gpmCO}_2)]}$$

F.E. = miles/gal

4. For our purposes, a fuel specific gravity must be assumed as well as a carbon/ hydrogen ratio. Also, grams/mi of exhaust components is unavailable as a direct measurement. The volume fraction of the exhaust constituents is available, however, and a "performance factor" form of the equation may be written in terms of volume percent. A more technically correct form of the equation would be based on mass fraction, which is discussed next.

II. P.F.= $\frac{3785 \text{ SG}}{(0.0833 \text{ x}+1)[\frac{1}{9}(0.0833 \text{ x}+1)(\frac{1}{9}(0.0833 \text{ x}+1))(\frac{1}{9}(0.0833 \text{ x}+1)(\frac{1}{9}(0.0833 \text{ x}+1))(\frac{1}{9}(0.0833 \text{ x}+1))($

(no meaning to units obtained)

PUSHINIG MGA 90 Emissions Tests Analysis Baseline Test Date View 72 82 Treated Test Date TIDEWLATEN MARNE Company Year and Make of Equipment Company Identification No. BRAHMA Gens Engine C.I.D. and Cylinders 2 PU-71 Alains Serial Number Precombustion // After cooled // Intercooled // Turbo // Natural Asp. / Baseline Ambient Temperature P27/Treated Ambient Temperature Catalytic Converter No Scrubber No Baseline Engine Temperature Treated Engine Temperature Baseline Fuel: Diesel #1____ #2 // #3___ Other _____ Treated Fuel: Diesel #1____ #2___ #3___ Other_____ Baseline Fuel: Premium Regular Unleaded Othen Treated Fuel: Premium Regular Unleaded Other Baseline Engine Hours or Mileage Treated Engine Hours or Mileage Treated MGA Calibrated_ Baseline MGA Calibrated STARGOARS MAY PATTOG Terres - Treste Treated 3 Baseline 2 Treated 1 Baseline 1 Rom 1700 3.9% co, 5.2% -5% 12% 0, 12% 72 HC .0502 CO STARSOARD RPM Baseline 1 Baseline 2 Freated * C0, 0, HC CO

MGA 90 Emissions Test Analysis Page 2

Por MAIN

0		POE P	CALN	
		-	Base	BASE
RPM	Baseline 1	Baseline 2	Treated 1	Treated 2
APAU 1720	23	1100 507	Contraction of the	1150
CO ₂			26-	_ S Gr
02	13.5	12.2%	11.57	11.5 00
HC	17	20	20	19
CO	,202	, 21%	- 150%	. 190 %

Por Gen

Average____

Average

Average____

Average_____

3

RAM COZ OZ HC / Co

MGA 90 Emissions Tests Analysis
Baseline Test Date Treated Test Date
Company TIDEWARD
Year and Make of Equipment
Company Identification No
Engine C.I.D. and Cylinders
Serial Number
Precombustion // After cooled // Intercooled // Turbo // Natural Asp. //
Baseline Ambient Temperature Treated Ambient Temperature
Catalytic Converter Scrubber
Baseline Engine Temperature Treated Engine Temperature
Baseline Fuel: Diesel #1#2#3Other
Treated Fuel: Diesel #1 #2 #3 Other
Baseline Fuel: Premium Regular Unleaded Other
Treated Fuel: Premium Regular Unleaded Other
Baseline Engine Hours or Mileage Treated Engine Hours or Mileage
Baseline MGA Calibrated Treated MGA Calibrated
RPM Baseline 1 Baseline 2 Treated 1 Treated 2 KPL_1 1500 1500 1500 1500 1500 1500 1500 1500 1500 1500 1500 1500 2.80 2.80 2.80 2.80 2.80 1000 1000 1000
RPMBaseline 1 Baseline 2 Treated 1 Treated 2
co ₂
0 ₂
НС
CO

MGA 90 Emissions Test Analysis Page 2

Por Hans

RPM-	Baseline 1	Baseline 2	Treated 1	Treated 2
co ₂	1500	2.8%	1500 3.1	
02	15.5%	15.57	15%	
HC	20	20	2.0	
CO	T	. 010	,040	
				1

Average Average Average

Average_____ Average_____ Average 1

Port Ger

002 02 HC CD

PULSITO MGA 90
Emissions Tests Analysis
Baseline Test Date Treated Test Date
Company TIDEWATEN
Year and Make of Equipment
Company Identification No
Engine C.I.D. and Cylinders
Serial Number
Precombustion // After cooled // Intercooled // Turbo // Natural Asp. //
Baseline Ambient Temperature Treated Ambient Temperature
Catalytic Converter Scrubber
Baseline Engine Temperature Treated Engine Temperature
Baseline Fuel: Diesel #1 #2 #3 Other
Treated Fuel: Diesel #1 #2 #3 Other
Baseline Fuel: Premium Regular Unleaded Other
Treated Fuel: Premium Regular Unleaded Other
Baseline Engine Hours or Mileage Treated Engine Hours or Mileage
Baseline MGA Calibrated Treated MGA Calibrated
RPM 1200 Baseline 1 Baseline 2 Freated 1 Treated 2
co2 2.27 2.4 2.4
02 17% 16.8 16.7
HC 18 21 19
co <u>,010</u> <u>,01</u> <u>,01</u> 1200 Port M Bare
RPM 128 Baseline 1 Baseline 2 Treated 1 Treated 2
co, <u>26 2.0%</u> 2.0
0, 17.97. 17.3
HC 20 18 19
co .070 .01 .0]

	MGA 90 Emissions Tests Analysis
	Baseline Test Date Treated Test Date
	Company TIDEWATER MARINE
	Year and Make of Equipment
	Company Identification No. BRAHMA
	Engine C.I.D. and Cylinders 2 - 8U-71 Mars 2- 471 Gens
	Serial Number
	Precombustion // After cooled // Intercooled // Turbo // Natural Asp. //
	Baseline Ambient Temperature Treated Ambient Temperature
	Catalytic Converter No Scrubber No
	Baseline Engine Temperature Treated Engine Temperature
	Baseline Fuel: Diesel #1 #2 #3 Other
	Treated Fuel: Diesel #1 #2 #3 Other
	Baseline Fuel: Premium Regular Unleaded Other
	Treated Fuel: PremiumRegularUnleadedOther
	Baseline Engine Hours or Mileage Treated Engine Hours or Mileage
	Baseline MGA Calibrated Treated MGA Calibrated
1700 Pu	RPM Baselin e 1 Baselin e 2 Treated 1 Treated 2
	co2 3.00% 2.98% 2.990
STARBOALD	02 13.5% 13.4% 13.4%
	HC 13.0 15.0 15.0
	co 0.07% 0.06% 0.05%
	RPM Baseline 1 Baseline 2 Treated 1 Treated 2
2	co2 <u>2.98</u> <u>2.9%</u> <u>2.9%</u>
PORT	02 13.5% 13.49% 13.25%
	HC 123 15.0 15
	co D.11% 0.15% 0.17%

No.

PORT

	MGA 90 Emissions Tests Analysis
	Baseline Test Date Treated Test Date
	Company TIDEWATER
	Year and Make of Equipment
	Company Identification No
i a	Engine C.I.D. and Cylinders
	Serial Number
	Precombustion // After cooled // Intercooled // Turbo // Natural Asp. //
	Baseline Ambient Temperature Treated Ambient Temperature
	Catalytic Converter Scrubber
	Baseline Engine Temperature Treated Engine Temperature
	Baseline Fuel: Diesel #1 #2 #3 Other
	Treated Fuel: Diesel #1 #2 #3 Other
	Baseline Fuel: PremiumRegularUnleadedOther
	Treated Fuel: Premium Regular Unleaded Other
	Baseline Engine Hours or Mileage Treated Engine Hours or Mileage
0	Baseline MGA Calibrated Treated MGA Calibrated
1500 Pu	RPM Baseline 1 Baseline 2 Treated 1 Treated 2
STARBOARD	$P_2 = \frac{14.99}{15.01} \frac{15.01}{15} \frac{1690}{15}$
	$co \qquad 0.7 \qquad$
	TREATES TREATES
	RPM Baseline 1 Baseline 2 Treated 1 Treated 2
Port	co2 <u>1.8% [.85%].84</u> 20
T OKT	02 15.01% 15.2% 15.6%
	HC 15.0 15.0 15.0
	$co \underline{O} \underline{b} \underline{O} \underline{w} \underline{O} \underline{w}$

	MGA 90 Emissions Tests Analysis	
	Baseline Test Date Treated Test Date	
	Company TIDEWATER	
	Year and Make of Equipment	
	Company Identification No	
	Engine C.I.D. and Cylinders	
	Serial Number	
	Precombustion // After cooled // Intercooled // Turbo // Natural Asp. //	7
	Baseline Ambient Temperature Treated Ambient Temperature	
	Catalytic Converter Scrubber	
	Baseline Engine Temperature Treated Engine Temperature	
	Baseline Fuel: Diesel #1 #2 #3 Other	
	Treated Fuel: Diesel #1#2#3Other	
	Baseline Fuel: PremiumRegularUnleadedOther	
	Treated Fuel: Premium Regular Unleaded Other	
	Baseline Engine Hours or Mileage Treated Engine Hours or Mileage	_
	Baseline MGA Calibrated Treated MGA Calibrated	
1200 Pus	RPM Baseline 1 Baseline 2 Treated 1 Treated 2	
	$co_2 1.2\% 1.05\% 1.0$	
STAR BOARD	02 17.5% 18.0% 18.2	
	HC 12.5 130 135	
	co <u>0%</u> <u>0%</u> <u></u>	
	RPM Baseline 1 Baseline 2 Treated 1 Treated 2	
	$co_2 1.170 1.170 1.09\%$	
Port	02 17.690 17.98 18%	
	HC 14.0 12.5 13	
	$co \underline{020} \underline{0} \underline{020} \underline{020}$	

LIPRINER	
MGA 90 Emissions Tests Analysis	1
Baseline Test Date Aug 2,1982 Treated Test	Date
Company IDEWATER MARINE	
Year and Make of Equipment	
Company Identification No. MN BRAAM	A
Engine C.I.D. and Cylinders	
Serial Number	
Precombustion // After cooled // Intercooled //	Turbo/7 Natural Asp./7
Baseline Ambient Temperature Treated Ambie	
Catalytic Converter Scrubber	
Baseline Engine Temperature Treated Engin	
Baseline Fuel: Diesel #1 #2 #3 Other	
Treated Fuel: Diesel #1 #2 #3 Other	
Baseline Fuel: Premium Regular Unleaded	
Treated Fuel: Premium Regular Unleaded	
Baseline Engine Hours or Mileage Treated Engi	A
Baseline MGA Calibrated Treated MGA	Calibrated
RPM 1000 Baseline 1 Baseline 2 Treated 1 Tr	reated 2
co2 3.25703.290	
02 14.9970 14.95%	
HC <u>700</u> <u>70</u>	
CO TRACE TRACE	
STBP DDM (7/2) Recelling 1 Recelling 2 Treated 1 Tr	control 2
$RPM_{1700} Baseline 1 Baseline 2 Treated 1 Tr CO2 \underline{3.19}_{2} 2.999_{0}$	
$0_2 $ <u>159k 15.19</u>	
HC <u>20</u> <u>20.05</u>	
CO TRACE TRACE	

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HPRINER	1
MGA 90 Emissions Tests Analysis	
Baseline Test Date / 469 22,1982 Treated Test Date	
Company TOEWATER MARINE	
Year and Make of Equipment	
Company Identification No. N/V BRAHMA	
Engine C.I.D. and Cylinders	
Serial Number	
Precombustion // After cooled // Intercooled // Turbo // Natura	al Asp. ///
Baseline Ambient Temperature Treated Ambient Temperature	
Catalytic Converter Scrubber	
Baseline Engine Temperature Treated Engine Temperature	1
Baseline Fuel: Diesel #1 #2 #3 Other	
Treated Fuel: Diesel #1 #2 #3 Other	
Baseline Fuel: Premium Regular Unleaded Other	
Treated Fuel: Premium Regular Unleaded Other	
Baseline Engine Hours or Mileage Treated Engine Hours or Mile	eage
Baseline MGA Calibrated Treated MGA Calibrated	
RPM Baseline 1 Baseline 2 Treated 1 Treated 2	
co2 2.21% 2.1.9%	
02 16.2590 16.890	1
HC 20.0 70.0	
STBD TRACE TRACE	
RPM 1 Baseline 1 Baseline 2 Treated 1 Treated 2	1 1
$co_2 1.9 1.890$	
02 16.8 A.M.	
HC 20.0 70.0	
CO TRACE TRACE	
· ·	1

	MGA 90 Emissions Tests Analysis
	Baseline Test Date Treated Test Date Auc 13
	Company TUDEWATER
	Year and Make of Equipment
	Company Identification No.
	Engine C.I.D. and Cylinders
	Serial Number
	Precombustion // After cooled // Intercooled // Turbo // Natural Asp. //
	Baseline Ambient Temperature Treated Ambient Temperature
	Catalytic Converter Scrubber
	Baseline Engine Temperature Treated Engine Temperature
	Baseline Fuel: Diesel #1#2#3Other
	Treated Fuel: Diesel #1 #2 #3 Other
	Baseline Fuel: PremiumRegularUnleadedOther
	Treated Fuel: PremiumRegularUnleadedOther
	Baseline Engine Hours or Mileage Treated Engine Hours or Mileage
1700	Baseline MGA Calibrated Treated MGA Calibrated

Baseline 1 Baseline 2 Treated 1 Treated 2 RPM 1.5% 1.58 1.45% CO2 17.4% 17.4% 17.2 02 12 9 НC % \bigcirc Õ СО 0 Baseline 1 Baseline 2 Treated 1 RPM Treated 2

1.2%

0%

11

18.0%

D

11.5

1.2 % 18.1%

((.0)

0.0%

STBD

Port

со₂

02

HC

СО

		\bigcirc		A 90 ests Analysi	s		
	Baseline	Test Date		Treated Te	est Date	5.	
	Company	TIDEWATE	\sim				
		Make of Equi					
		Identification					
		.I.D. and Cyl					
	Serial Nu	mber					
	Precombu	stion / Afte	r cooled/7	Intercooled /	7 Turbo/7	Natural	Asp. 🗂
		Ambient Temp					
	Catalytic	Converter		Scrubber	No. of Concession, State of Concession, State		
	Baseline	Engine Temper	rature	Treated Er	igine Tempera	ature	
	Baseline	Fuel: Diesel	#1#2	#3 Othe	er		
	Treated I	-uel: Diesel #	±1 #2	#3 Othe	r		-
	Baseline	Fuel: Premiur	m Regula	unlead	ed Other		
	Treated H	-uel: Premium	n Regular	Unleade	ed Other_		
	Baseline I	Engine Hours	or Mileage	Treated E	ingine Hours	or Milea	ge
1500	Baseline I	MGA Calibrate	d	Treated M	GA Calibrated	1	
	RPM	Baseline 1	Baseline 2	Treated 1	Treated 2		
	CO ₂	0.87%	1.01%				
Por	0 ₂	19.090	18.7%			٨	
	НС	11.0	11.0				
	СО	090	0_10				
	RPM	Baseline 1	Baseline 2	Treated 1	Treated 2		P
	CO,	0.890					
STBD	0,	19.7%					
	HC	11.0					
	CO	0%				. 1	
		anna Stan an Ani					

. .	DOULRRIVED	
		GA 90 Tests Analysis
	Baseline Test Date Juin 22,198	C Treated Test Date
	Company TIDEWATER	
	Year and Make of Equipment	
	Company Identification No.	V BRAHMA
	Engine C.I.D. and Cylinders	
	Serial Number	
		Intercooled / Turbo / Natural Asp. /
	Baseline Ambient Temperature	Treated Ambient Temperature
	Catalytic Converter	Scrubber
	Baseline Engine Temperature	Treated Engine Temperature
	Baseline Fuel: Diesel #1 #2	#3 Other
	Treated Fuel: Diesel #1 #2	#3 Other
	Baseline Fuel: Premium Regul	ar Unleaded Other
	Treated Fuel: Premium Regula	ar Unleaded Other
	Baseline Engine Hours or Mileage_	Treated Engine Hours or Mileage
		Treated MGA Calibrated
P34	×	BASELINE 3
	$\begin{array}{c} \text{RPM} \boxed{120} \text{ Baseline 1} & \text{Baseline 2} \\ \text{CO}_2 & \boxed{2.8\%} & \boxed{3.0\%} \end{array}$	Treated 1 Treated 2
	0, 15.6% 15.2%	
	HC 70.0 70.0	
	CO TRACE TRACE	· · · · · · · · · · · · · · · · · · ·
-571-		BADICLINE 3
	RPM Baseline 1 Baseline 2	Treated 1 Treated 2
	co2 <u>1.61%</u> <u>2.966</u>	
	02 15.5% 15.5k	
	HC 20.0 700	
	CO TRACE TRACE	

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,	DOLLIPEI	VER						
				A 90 ests Analysi	s			
	Baseline T	est Date ALK	V 22, 1982	Treated Te	est Date			
		Make of <mark>E</mark> qui						
	Company I	Identification	No. M	I BRAH	AMA			
						Λ		
	Serial Num	nber						
	Precombus	tion // Afte	r cooled/_7	Intercooled /	7 Turbo/7 N	atural Asp./_/		
	Baseline A	mbient Temp	erature	Treated A	mbient Tempera	ture		
	Catalytic (Converter		Scrubber				
	Baseline Engine Temperature Treated Engine Temperature							
	Baseline F	uel: Diesel	#1#2	#3 Othe	er			
	Treated Fu	uel: Diesel #	#1#2	#3 Othe	r			
	Baseline F	uel: Premiur	m Regula	r Unlead	edOther			
	Treated Fu	uel: Premiun	n Regular	Unleade	edOther			
	Baseline E	ngine Hours	or Mileage	Treated E	ngine Hours or	· Mileage		
R	Baseline M	GA Calibrate	d	Treated M	GA Calibrated_			
	RPMise	Baseline 1	Baseline 2	Treated 1	Treated 2			
	co ₂	2.2%	2.4%					
	0 ₂	16.9%	Ke. Vo		х.			
	HC	19.0	10.0		-			
	СО	TRACE	TRACE			A		
ALX.) RPM <u>//3</u> 00	Baseline 1	Baseline 2	Treated 1	Treated 2			
	co ₂	2.1/0	2:4/0					
	02	16.67	16.270					
	НС	20,0	19.0					
	CO	TRAE	TRACE					

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DC	KIARNER	١	
	MGA 90 Emissions Tests Analysis		
	Baseline Test Date ALY 72, 1982 Treated Test Date		
	Company TIDEWATER MARINE		
	Year and Make of Equipment	1	
	Company Identification No. M/V BRAHMA		
	Engine C.I.D. and Cylinders		
	Serial Number		
	Precombustion / After cooled / Intercooled / Turbo / Natura	I As	р./7
	Baseline Ambient Temperature Treated Ambient Temperature		<u> </u>
	Catalytic Converter Scrubber	X	
	Baseline Engine Temperature Treated Engine Temperature		
	Baseline Fuel: Diesel #1 #2 #3 Other		
	Treated Fuel: Diesel #1 #2 #3 Other		
	Baseline Fuel: Premium Regular Unleaded Other		
	Treated Fuel: Premium Regular Unleaded Other		
	Baseline Engine Hours or Mileage Treated Engine Hours or Mile	age	
	Baseline MGA Calibrated Treated MGA Calibrated		
	RPM	ġ.	
	$co_2 1.6170 1.520$		
	$0_2 17.59_0 17.79_0$		
	HC 1900 19.0		
	co TRACE TRACE		ſ
	STBO RPM <u>IZ00</u> Baseline 1 Baseline 2 Treated 1 Treated 2	A	1
	co2 1.62.6 1.792		
	02 17.490 17.3%	1	
	HC 18.0 18.58		
	CO TRACE TRACE		

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		IGA 90 Tests Analysis
	Baseline Test Date	Treated Test Date Auce. 13
	Company TIDEWATER	
	Year and Make of Equipment	N
	Company Identification No	
	Engine C.I.D. and Cylinders	
	Serial Number	A
	Precombustion // After cooled //	Intercooled / Turbo / Natural Asp. /
	Baseline Ambient Temperature	Treated Ambient Temperature
	Catalytic Converter	Scrubber
	Baseline Engine Temperature	Treated Engine Temperature
	Baseline Fuel: Diesel #1 #2	#3 Other
	Treated Fuel: Diesel #1 #2	#3Other
	Baseline Fuel: Premium Regu	lar Unleaded Other
	Treated Fuel: Premium Regul	arUnleadedOther
	Baseline Engine Hours or Mileage_	Treated Engine Hours or Mileage
D		Treated MGA Calibrated
1700 Da	TREMED TREATED	Treated 1 Treated 2
Λ	co2 1.59% 1.5%	1.590
PORI .	02 17.4% 17.29	6 17.490
	нс <u>11.0</u> <u>11.0</u>	11.0
	co oh oh	090
	RPMBaseline 1 Baseline 2	Treated 1 Treated 2
	co2 1.3 1.289	1.3.90
STARBOARD	0 ₂ <u>17.9</u> <u>17.8</u> °	0 17.8%
	HC 11.0 11.0	10.8
	co <u>0</u> <u>0</u> <u>7</u> 0	<u> 0%</u>

		\bigcirc	MGA Emissions Te	90 A 90 Sts Analysis			<u>*</u>
	Baseline Tes	st Date		Treated Te	st Date Au	16.13	
	Year and Ma	ake of Equip	oment				
	Company Id	en <mark>t</mark> ification	No				
	Engine C.I.	D. and Cyli	nders			1 1	
	Serial Numb	er					
	Precombusti	on// After	cooled/7	ntercooled/	7 Turbo/7 N	Natural A	sp.∏
	Baseline Am	bient Tempe	erature	_ Treated An	nbient Temper	ature	
	Catalytic Co	onverter		Scrubber		,	
	Baseline Eng	gine Temper	ature	Treated En	gine Temperat	ture	
	Baseline Fu	el: Diesel #	±1 #2	#3 Othe	r		Í
	Treated Fue	el: Diesel #	1 #2	#3 Other	ſ		
	Baseline Fu	el: Premiun	n Regular	Unleade	edOther	· ·	
	Treated Fue	el: Premium	Regular	Unleade	dOther	· · · · ·	
	Baseline Eng	gine Hours o	or Mileage	Treated E	ngine Hours d	or Mileage	ə
17			£	Treated MG	GA Calibrated		
Nou	RPM I		<i>Theareo</i> B aselin e 2	Treated 1	Treated 2		
	C0,	1.0 %	1.0%	1.10%			
	0,	18.6%	18,6%	18.5%			
	HC	10,8	11.0	_11.0			
	СО	ON	0%	0%		1	
	RPM	R <i>emes</i> Baseline 1	<i>Theres</i> Baseline 2	Treated 1	Treated 2		1 .
	co ₂	0.87%	6.90	0.80%			
	0 ₂	19.0%	18.8	19.059	υ		
	НС	N.O	10.5	11.0			
	CO	8%	0%	0%		1	

Poet

1500

STBD

	MGA 90 Emissions Tests Anal	ysis											
	Baseline Test Date Treated	Test Date											
	Company TIDEWATER	· · · · · · · · · · · · · · · · · · ·											
	Year and Make of Equipment												
	Company Identification No												
	Engine C.I.D. and Cylinders												
	Serial Number												
	Precombustion // After cooled // Intercoole	d/7 Turbo/7 Natural Asp./7											
	Baseline Ambient Temperature Treated	Ambient Temperature											
	Catalytic Converter Scrubb	er											
	Baseline Engine Temperature Treated	Engine Temperature											
	Baseline Fuel: Diesel #1 #2 #3 O	ther											
	Treated Fuel: Diesel #1 #2 #3 0	ther											
	Baseline Fuel: PremiumRegularUnle	eadedOther											
	Treated Fuel: Premium Regular Unle	adedOther											
	Baseline Engine Hours or Mileage Treate	d Engine Hours or Mileage											
Daus	Baseline MGA Calibrated Treated	MGA Calibrated											
<i>C c c c c c c c c c c</i>	RPMBaseline 1 Baseline 2 Treated	1 Treated 2											
	$c_{0}, 0.7\% 6.779, 0.79$	9											
	0, 19.490 10.29 19.2	90											
	HC 11.0 12.0	· 1											
	co O O O O O O O O O	6											
	TREATES TREATES												
	RPMBaseline 1 Baseline 2 Treated	1 Treated 2											
	$co_2 \underline{0.610} \underline{0.67} 0.67$	-9											
	$0_2 \qquad 10.590 10.50 10.50$	> <i>W</i>											
	$\begin{array}{cccccccccccccccccccccccccccccccccccc$												
	co												

Port

1200

STBD

R.P.M.	Up Ri Port		Up Ri Stbo			n River ort		n River tbd	Pu Po	shing rt		Pusl Stbo			Gene Stb	erator d	R.P.M. Fixed	Emissions Level
	1 2	2	1	2	1	2	1	2	1	2	3	1	2	3	1	2	3	
1200	1.1 1	1.6	1.18	1.7	1.61	1.5	1.62	1.7	2.0	2.0	2.0	2.2	2.4	2.4	1.8	1.9	1.9	CO ₂
	18.5 1	17.49	18.25	17.75	17.5	17.7	17.4	17.3	17.4	17.3	17.3	17	16.8	16.7	17.0	17.0	17.0	02
	19 2	20	20	20	19.	19	18	18.5	20	18	19,	18	21	19	20.0	125	25	HC
	Tr 7	Γr	Tr	Tr	Tr	Tr	Tr	Tr	.010	.10	.010	.01	.01	.01	Tr	Tr	Tr	CO
	1 2	2	1	2	1	2	1	2	1	2	3	1	2	3				
1500	2.21	1.9	1.9	1.8	2.2	2.4	2.1	2.4	3.0	2.8	3.1	3.8	2.8	2.9				CO ₂
	16.25		16.8	17	16.9	16.1	16.6	16.2	15.5	15.5	15	14	15.5	15.5				02
	20	20	20	20	19.	19	20	19	20	20	20	20	20	20				ΗĈ
	Tr	Tr	Tr	Tr	Tr	Tr	Tr	Tr	Tr	.01	.02	.05	.01	.01				CO
	1 2	2	1	2	1	2	1	2	1	2	3	4	1	2	3	4		
1700	3.25	3.2	3.1	2.99	2.8	3.0	1.61	2.96	2.3	5.0	5.0	5.0	3.9	5.2	5.0	4.8		CO ₂
	14.78	14.95	15	15.1	15.6	15.2	15.5	15.5	13.5	12.2	11.5	11.5	12	12	12	12		0 ₂
	20	20	20	20.05	20	20	20	20	17	20	20	19	22	22	21	21		HC
	Tr	Tr	Tr	Tr	Tr	Tr	Tr	Tr	. 20	.21	.15	.19	.05	.06	.05	.05		СО

Tidewater Marine Baseline Evaluation

File in Tidewater file